## 337413(37)

# B. E. (Fourth Semester) Examination, April-May 2021

(Old Scheme)

(Mech. Engg. Branch)

### APPLIED THERMODYNAMICS

Time Allowed: Three hours

Maximum Marks: 80

Minimum Pass Marks: 28

Note: Part (a) of each unit is compulsory and solve any two from (b), (c) and (d). Use of Steam table and Mollier diagram is printed.

## **Unit-I**

1. (a) Define Entropy principle?

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b)	Explain the equivalence of Kelvin planck and	clausius	
	statement		

(c) A heat engine produces work equivalent to 80 kW with an efficiency of 40%. Determine the heat transfer rate to and from the working fluid.

(d) A heat engine develops 10 kW power when receiving heat at the rate of 2250 kJ/min. Evaluate the corresponding rate of heat rejection from the engine and its thermal efficiency.

- 2. (a) Define law of corresponding state.
  - (b) Derive first T-ds equation.
  - (c) Setup the Tds relation.

$$Tds = c_{p}dt - \beta v T d\rho$$

Where symbols have their usual meaning.

(d) Derive the following reduced equation of state.

$$\left(P_r + \frac{3}{v_r^2}\right)(3v_r - 1) = 8T_r$$

## become notice of the Unit-III to the the transfer of the

3.	(a) Define pure substances.
	(b) Steam at 10 bar pressure and 0.9 deryness fraction
	is cooled at constant volume at 160°C. What will
	be the final condition.

(c) Find the enthalpy, entropy and volume of steam at 1.4 MPa, 380°C.

(d) A vaccum of 710 mm was obtained in a condenser when the barometer reads 755 mm. The temperature of condensate was 25°C. Determine the pressure of air and steam in the condenser and the mass of air per kg of steam. Also determine the volume efficiency.

- 4. (a) Draw P-V and T-S diagram for reversed Carnot cycle.
  - (b) Explain the simple vapour compression refrigeration cycle.
  - (c) Derive the equation for volumetric efficiency of air compressor.
  - (d) A single acting two stage air compressor deals with

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4 m<sup>3</sup>/min of air at 1.013 bar and 15°C with a speed of 250 rpm. The delivery pressure is 80 bar. Assuming complete intercooling, find the minimum power required by the compressor and the bore and stroke of the compressor. Assume a piston speed of 3 m/s. Mechanical efficiency of 75% and volumetric efficiency of 80% per stage. Assume polytropic index of compression in both the stages to be n = 1.25 and neglect clearance.

(a) Define Mach number. 5.

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(b) Derive the following equation.

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$$\frac{dA}{A} = \left(M^2 - 1\right) \frac{dV}{V}$$

(c) Derive the following equation.

$$\frac{dA}{A} = \frac{\left(M^2 - 1\right)dm}{m\left(1 + \frac{Y - 1}{2}M^2\right)}$$

$$\lim_{t \to \infty} \frac{T_o}{T} = 1 + \frac{Y - 1}{2} M^2$$